# ASSESSING THE INFLUENCE OF THE GLOBAL EMISSIONS REDUCTION TARGETS ON THE DESIGN AND PERFORMANCES OF A SHIP'S PROPULSION SYSTEM

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### **ABSTRACT**

The paper presents aspects related the influence of the global gases emissions reduction targets on the design and performances of a propulsion system for a bulk carrier. In a first stage, the key elements of the ship propulsion system: main engine and propeller have been chosen and designed. The aim was to obtain the maximum propulsive efficiency at the owner desired speed. Different combinations: diesel engine-optimal propeller have been designed and analysed from the efficiency point of view. In a second stage, considering the mandatory IMO regulations concerning CO2 emissions, the EEDI (Energy Efficiency Design Index) has been computed for every designed case. To compliance EEDI targets, it was been necessary to reduce the ship speed, and a new propulsion system with lower speed performances has been designed. In this study, ship hull shapes were given and EEDI became an important and critical factor in the design of the ship propulsion system. Other potential pathways to achieve emissions reduction targets will need to be identify in a future study.

Keywords: ship propulsion system, Energy Efficiency Desin Index EEDI

### 1. INTRODUCTION

The shipping industry has an important role in the international trade, but also contribute to global greenhouse gases emissions, accounting for approximately 3% of the total. To address this challenge, IMO (International Maritime Organization) has introduced technical and operational measures to reduce ships emissions [1]. As technical measures the EEDI (Energy Efficiency Design Index) has become mandatory for new constructed ships, and all existing ships must comply with EEXI (Energy Efficiency Existing Ship Index) requirements. SEEMP (Ship Energy Efficiency

Management Plan) and CII (Carbon Intensity Indicator) are related to operational measures.

EEDI represents a technical measure to reduce Green House Gases (GHG) emissions from ships applied for new ships and its compliance is evaluated in the design stage. Thus, EEDI becomes an important tool in the propulsion system design for the new, more energy efficient and less pollutant ships.

The paper presents a study focused on the influence of the global gases emissions reduction targets on the design and performances of a propulsion system for a 28000 tdw bulk carrier. The work was initially carried out for the preparation of a bachelor's thesis and then it has been developed to add new research data

on the topic of reducing gas emissions for different types of ships and capacities. Such studies have also performed in the previous years in the Centre of Research of the Naval Architecture Faculty of "Dunarea de Jos" University in Galati.

The main geometrical characteristics of the ship are given in Table 1 and the hull shapes are plotted in Figure 1.

**Table 1**. Ship main dimensions

| Lenght over all  | 175,9 | [m] |
|------------------|-------|-----|
| Length waterline | 166,3 | [m] |
| Beam             | 28    | [m] |
| Draught          | 9,5   | [m] |

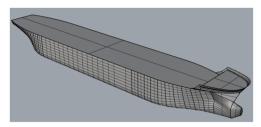


Fig.1 Ship hull shapes

In a first stage, the key elements of the ship propulsion system: main engine and propeller have been chosen and designed. The aim was to obtain the maximum propulsive efficiency at the owner required speed. Usually, for bulk carrier, slow speed two stroke Diesel engine direct coupled with Fixed Pitch Propellers are used to propel the ships speeds range 12-15 knots.

Different combinations: diesel engineoptimal propeller have been analysed from the efficiency point of view. In this stage, the investigation into propulsion efficiency has been performed in eight cases: four selectrd diesel engines, each engine with two propellers designed and optimized for different operational conditions (two different propellers design points).

In a next stage, the mandatory IMO regulations concerning CO2 emissions have been taken into account, and the EEDI (Energy Efficiency Design Index) has been computed for every designed case. Noting that, from the

perspective of EEDI, the desired speed was not feasible, to compliance EEDI targets, it was been necessary to reduce the ship speed, and a ninth one case has been analysed, a new propulsion system with lower ship speed performances has been designed.

In this study, ship hull shapes were given and EEDI became an important and critical factor in the design of the ship propulsion system. Other potential pathways to achieve emissions reduction targets will need to be identify in future research. The study has shown one more time that the relationship between ship propulsion system and EEDI regulatory targets is a complex and interesting challenge for naval architects and designers. On the other hand, EEDI allows ship owners to opt for the most economically and convenient technological solutions to achieve the emission reduction objectives [2].

# 2. SHIP'S PROPULSION PERFORMANCES ASSESSMENT

In the first stage of the present work, preliminary design of the propulsion system for the given deadweight bulk carrier has been performed and the propulsive performances for different combinations diesel engine – optimal efficiency propeller have been analyses. For this purpose, initially, the total hydrodynamic ship resistance has been carried out using empirical methods such as Holtrop-Mennen approach. The results for a range of ship velocity around the owner desired speed have been plotted in diagram from Figure 2.

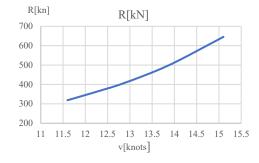


Fig. 2. Hydrodynamic ship resistance

These results have been used as initial data for necessary propulsion power computation, in order to select the appropriate main engine for this vessel. Four two stroke slow speed Diesel engines were selected, and for each main engine, two propellers were designed at different design points, resulting in eight analysis cases.

Usually, in propeller design, two propulsion margines are added to the design point (PD): SM-Sea Margin and EM-Engine Margin (Figure 3 [3]).

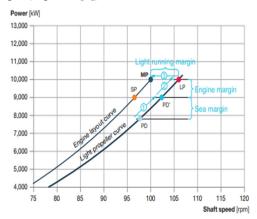


Fig. 3. Example for Propeller design points selection [3]

Sea Margin (SM) takes into account the increase of the ship resistance caused by wind, waves, hull fouling, ensuring the ship maintains design speed under these conditions. SM is common estimated at 15%. The propeller is often designed to the alternative design point PD' including SM. Engine Margin (EM) is recommended for economical operation, lower fuel consumption and to have a power reserve for increased speed and it is typically around 10%. A trend in increasing the Engine Margine has been observed to meet IMO requirement regarding ships emission reduction.

In the present study, these propulsion margins were introduced in a so named power utilization coefficient (cu). For each selected engine, two optimal propellers were designed to consume the delivered power in two design points, for cu=0.85 and cu=0.75. Diagrams for B Wageningen series were used to find the optimum efficiency propellers (Ŋo – open water efficiency) and their geometry (diameter-D, number of blades-z, pitch ratio-P/D, blade area ratio Ae/Ao) to achieve the required ship speed V. The characteristics and performances of the designed ship propulsion systems are presented in Table 2 in report with ship's speed performances.

|           |             | Case1      | Case2 | Case3     | Case4 | Case5         | Case6 | Case7    | Case8 |
|-----------|-------------|------------|-------|-----------|-------|---------------|-------|----------|-------|
|           |             | MAN B&W    |       | WARTSILLA |       | WARTSILLA RT- |       | MAN B&W  |       |
| Engine    |             | S46ME C8.6 |       | RTA48T    |       | flex50        |       | G40ME B9 |       |
|           | Power[kw]   | 6300       |       | 6990      |       | 6100          |       | 6510     |       |
|           | Speed [rpm] | 105        |       | 102       |       | 99            |       | 106      |       |
|           | No.cyl.     | 7          |       | 6         |       | 5             |       | 7        |       |
|           | SFC[g/kWh]  | 163        |       | 171       |       | 167           |       | 171      |       |
| Propeller | cu          | 0,75       | 0,85  | 0,75      | 0,85  | 0,75          | 0,85  | 0,75     | 0,85  |
|           | D [m]       | 5,873      | 6,09  | 6,06      | 6,08  | 5,922         | 6,02  | 6,09     | 6,16  |
|           | Z           | 4          | 4     | 4         | 4     | 4             | 4     | 4        | 4     |
|           | Ae/Ao       | 0,55       | 0,55  | 0,55      | 0,55  | 0,55          | 0,55  | 0,55     | 0,55  |
|           | P/D         | 0,679      | 0,737 | 0,756     | 0,755 | 0,799         | 0,756 | 0,682    | 0,670 |
|           | По          | 0,57       | 0,58  | 0,56      | 0,58  | 0,58          | 0,59  | 0,57     | 0,58  |
| Ship      | V[knots]    | 13,56      | 13,97 | 13,96     | 14,46 | 13,44         | 13,91 | 13,67    | 14,12 |

**Table 2**. Ship propulsion systems characteristics and performances

In the first stage, the aim of the study was to design a propulsion system to achieve a velocity speed around 14.1 knots  $\pm$  0.2. From the

velocity required point of view, good results have been obtained in cases 2,3,6 and 8, respectively in the cases of engines with lower powers and cu=0.85, and for the engine with highest power and cu=0.75. It must be taken into account that, power utilization coefficient cu =0.75 (including SM=15% and EM=10%) is usually recommended for bulk carrier propellers design and in the recent decades, an increasing in EM is practiced to fulfil the EEDI objectives.

For each selected engine, by using a power utilization coefficient cu=0.75, a speed decrease around 0.5 knots (3%) was obtained in comparison to the design cases with cu=0.85. Related to optimal propeller performances, good values of the open water efficiency (0.56-0,59) were obtained for a Bulc carrier. The placement of an optimal propeller of maximum diameter was possible in all analysed cases.

In the following stage of this research, the results related ship propulsive efficiency were assessed in the context of Energy Efficiency Design Index requirements, analysing the impact of EEDI targets on the propulsion system design for the given bulk carrier.

# 3. ASSESSING SHIP PROPULSIVE PERFORMANCES IN THE CONTEX OF EEDI TARGETS

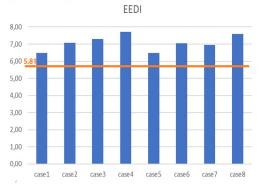
EEDI represents a technical measure introduced by IMO to reduce gases emissions from new ships. Its compliance is evaluated in the design stage, becoming a benchmark and a real challenge for the ship designer. The relationship between ship propulsion system and EEDI requirements is complex, the propulsive performances influencing energy and fuel consumption and implicit the CO<sub>2</sub> emissions from ships. EEDI is defined as the ratio between CO<sub>2</sub> emissions and transport work. An attained EEDI may be computed and compare with a required EEDI based on ship type and size, with emission reduction targets phased in over the years [4].

For attained EEDI computation, the following simplified formula may be used, based on power, fuel type and consumption, ship capacity and speed. Formula is more complex, including specific correction factors and coefficients.

$$EEDI = \frac{Power \ x \ CF \ x \ SFC}{Capacity \ x \ Speed} \tag{1}$$

The main engines propulsive power and auxiliary engines power too are taking into account in EEDI calculation, as well as the CO<sub>2</sub> conversion factor (CF) depending on type of fuel used and the Specific Fuel Consumption (SFC) as a characteristic of the engine. The transport work from the denominator is given by ship capacity defined usually function of ship deadweight multiplied with ship speed obtained in specified condition.

In a second stage of the present study, the mandatory IMO regulations concerning CO2 emissions have been taken into account for the designed ship propulsion systems, and the EEDI has been computed for every designed case. The results related attained EEDI in comparison with the required EEDI corresponding to Phase 2 (plotted as a line) are given in Figure 5.



**Fig.5.** Study cases results - attained EEDI versus required EEDI

It was found that, from the perspective of the Energy Efficiency Design Index, the speed of 14.1 knots is not feasible, none of the analysed cases comply with the CO<sub>2</sub> emissions targets. Therefore, the study continued in order to find solutions that would lead to EEDI's reduction, in order to fall within the allowed limits.

If we analyse EEDI formula only from mathematical point of view, as a fraction, in order to decrease its value (Figure 6) it would be necessary to decrease the numerator and increase the denominator.



**Fig.6.** EEDI formula as a mathematical fraction

But, taking into consideration the powerspeed cube law, this mathematical rule does not seem very easy to applied in ship propulsion system design, EEDI reduction becoming a real challenge for a naval architect. By increasing the denominator: respectively ship's capacity and speed, high propulsive power will be necessary, fact that can lead to denominator's rising, unfavourable situation from mathematical point of view.

During the last years, various technological innovations, strategies and factors with impact on ship propulsion design were explored to reduce EEDI: ship hull optimization, engine technology, alternative fuels, propeller efficiency, energy-saving devices.

It should be note that the ship hull was given and the engines were not chosen from the latest catalogues of the engine manufactures. Thus, the influence of some factors such as improvement in engine technology or dual fuel using, were not taking into account and these may be subject of a future work.

In the present study, the solution to reduce fuel consumption and CO2 emissions was to decrease ship's speed and power. Thus, the question arised as how much the power and ship speed should be reduced to ensure that the attained EEDI is within the admissible limits. For ship speed and power reduction, cases 1 and 5 were analysed, corresponding to the propulsion systems with the lowest power engines and optimal propeller designed at cu=0.75 design point. In these cases, the difference between the attained EEDI and the required EEDI are 11.611% (case1) and 11.6%

(case5) respectively. The new ship propulsion performances were estimated by a step by step decreasing of the ship speed and power. In a first step, 0.5 knots reduction of the ship's speed and of around of 11-12% in the propeller delivered power led to a reduction of the difference between required EEDI and attained EEDI between 1.84% and 2.2%. In a second step, for case 1, a new 0.5 knots reduction of the ship's speed and by about 22% in the delivered power to propeller led to an attained EEDI = 5,498 with 5,5% lower than the required EEDI = 5,819.

The propulsive performances and EEDI results after power-speed decreasing for case 1 are presented in Table 3. Delivered power to propeller was calculated function of engine power, shaft efficiency and power utilization coefficient cu.

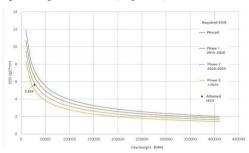
**Table 3**. EEDI versus ship propulsion system performances after power-speed reduction

| MAN B&W S46ME C8.6 |           |        | cu   | SM   | EM   |
|--------------------|-----------|--------|------|------|------|
|                    | MCR[kW]   | 6300   |      |      |      |
| Casela             | PD [kW]   | 4536   | 0,75 | 0,15 | 0,1  |
|                    | V [knots] | 13,56  |      |      |      |
|                    | EEDI      | 6,495  |      |      |      |
| Case1b             | PD [kW]   | 3977,1 | 0,66 | 0,15 | 0,19 |
|                    | V [knots] | 13,1   |      |      |      |
|                    | EEDI      | 5,92   |      |      |      |
| Caselc             | PD [kW]   | 3528,8 | 0,58 | 0,15 | 0,27 |
|                    | V [knots] | 12,6   |      |      |      |
|                    | EEDI      | 5,498  |      |      |      |

The results shown that, for the study case 1, selected for the present analysis due to reduced engine power and lower specifical fuel consumption, it was necessary to increase Engine Margin from 10% to 27% to meet EEDI requirements. In practice, an engine margin of 10-15% is recommended. Using a 20-30% Engine Margin is not the most efficient practice in normal operational conditions, leading to inefficiency in fuel consumption and higher operational costs,

Finaly, in the present study, the solution to reduce fuel consumption and CO2 emissions was to adopt lower ship speed, to select

another engine with lower power and to design a new optimal propeller. A diesel engine MAN B&W S40ME-C9.5, with a power of 4860 kW, 104 rpm and 6 cylinders was chosen. The geometrical characteristics of the optimal efficiency propeller designed at cu=0.75 design point, were: diameter D=5,8m, pitch ratio P/D=0,68. The resulted ship speed was 12,6 knots. The computed EEDI value for this new ship propulsion system was 5,62, lower with 3,55% than the required EEDI corresponding to Phase 2 (Figure 7).



**Fig.7** Attained EEDI versus Reference EEDI for the new propulsion system

### 4. CONCLUDING REMARKS

The paper presents a study focused on the influence of the global gases emissions reduction targets on the design and performances of a propulsion system for a 28000 tdw bulk carrier. In a first stage, the key elements of the ship propulsion system: main engine and propeller have been chosen and designed. The aim was to obtain the maximum propulsive efficiency at the owner required speed 14.1 knots  $\pm$  0.2. After hydrodynamic ship's resistance and necessary propulsive power estimation, four two stroke Diesel engines were selected, and for each engine, two propellers were designed at different design points, resulting in eight analysis cases.

In a second stage, the results related the ship propulsive efficiency were assessed in the context of Energy Efficiency Design Index requirements. The EEDI was computed for every study case, and it was found that, from the perspective of EEDI, the speed of 14.1

knots was not feasible, none of the analysed cases comply with the CO<sub>2</sub> emissions targets.

The adopted solution to reduce fuel consumption and CO2 emissions was to decrease ship speed and power. Initially it was estimated how much the ship's speed and engine power should be reduced in order to fulfil the EEDI requirements. Finally, the solution was to adopt a lower ship speed 12,6 knots, to select another engine with lower power and to design a new optimal propeller.

The work was initially carried out for the preparation of a bachelor's thesis and then it has been developed to add new research data on the topic of reducing gases emissions for different types of ships and capacities. Other potential pathways to achieve emissions reduction targets will need to be identify in a future study.

### Acknowledgements

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